

Mechanical equivalent of heat



Physics

Thermodynamics

Heat energy, thermal capacity

Applied Science

Engineering

Renewable Energy

Basic Principles

Applied Science

Engineering

Photonics

Basic Principles



Difficulty level

easy



Group size

1



Preparation time

10 minutes



Execution time

10 minutes

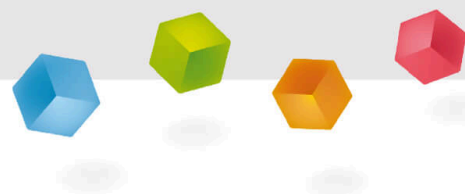
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General information



Application

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Fig.1: Experimental set-up

An understanding about the mechanical equivalent of heat is fundamentally important for the energy industry, as the heating of gases and using their mechanical equivalent of heat is used to produce electricity.

This experiment can be used to gain a first understanding.

Other information (1/2)

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**Prior****knowledge****Main****principle**

The prior knowledge required for this experiment is found in the theory section.

In this experiment, a metal test body is rotated and heated by the friction due to a tensed band of synthetic material. The mechanical equivalent of heat for problem 1 is determined from the defined mechanical work and from the thermal energy increase deduced from the increase of temperature. Assuming the equivalence of mechanical work and heat, the specific thermal capacity of aluminium and brass is determined.

Other information (2/2)

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**Learning
objective****Tasks**

The goal of this experiment is to investigate the mechanical equivalent of heat.

1. Determination of the mechanical equivalent of heat.
2. Determination of the specific thermal capacity of aluminium and brass.

Theory (1/3)

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For a long time it was debated as to whether the heat of a system, which used to be defined in terms of its temperature, was a form of energy or an independent magnitude which is conserved. During the first half of the nineteenth century, it was proven that mechanical energy due to friction is completely converted to heat, independently of the course of the transformation process and of the physical and chemical characteristics of the material used. Heat was accordingly defined as the energy of disorganized, macroscopically invisible molecular movements.

The quotient between realized mechanical work ΔW and the quantity of heat ΔQ generated through friction is called the mechanical equivalent of heat. In this experiment, mechanical work is performed by rotating the friction cylinder against the sliding frictional force F_R of the synthetic friction band. As weight M (cf. Fig. 2) is not accelerated when the crank is rotated, weight $F_G = Mg = 49 \text{ N}$ (M : mass of the weight, g : terrestrial gravitational acceleration) on one side and the sliding frictional force F_R acting together with force F_D on the suspension of the dynamometer on the other side, must cancel each other:

$$F_R = F_G - F_D \quad (1)$$

Theory (2/3)

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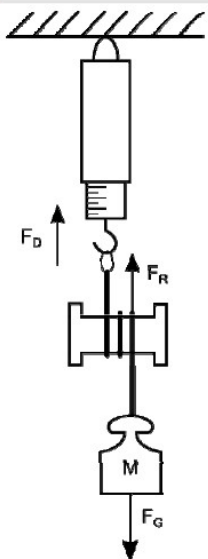


Fig. 2: Equilibrium of forces when the friction cylinder is rotated.

Friction work W is thus determined from the friction path of $2\pi r n$ (r : radius of the cylinder, n : number of turns) and is found to be:

$$W = 2\pi r n F_R = 2\pi r n (F_G - F_D) \quad (2)$$

Whilst the synthetic band slides over the cylinder, a large portion of mechanical work is converted to thermal energy ΔQ through inelastic deformations of the friction material. Accordingly, the temperature T of the friction cylinder increases by a temperature difference ΔT :

$$\Delta Q = C_{\text{tot}} \cdot \Delta T \quad (3)$$

Theory (3/3)

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C_{tot} is the total thermal capacity of the heated parts. It is the sum of the thermal capacity C_{cyl} of the friction cylinder (m: mass, specific thermal capacity (CuZn: $c = 0.385 \text{ J / gK}$)):

$$C_{cyl} = c \cdot m \quad (4)$$

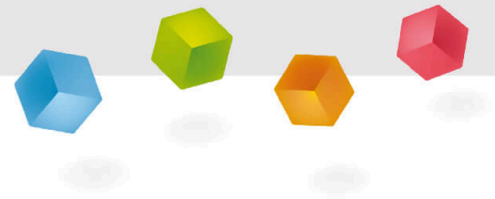
Of the thermal capacity of the friction band ($C_{band} = 4 \text{ J/K}$) and of the thermal capacity of the thermometer ($C_{th} = 4 \text{ J/K}$). If the mass of the friction cylinder is $m = 640 \text{ g}$, total thermal capacity amounts to:

$$C_{tot} = C_{cyl} + C_{band} + C_{th} = 254 \text{ J/K} \quad (5)$$

Equipment

Position	Material	Item No.	Quantity
1	Mechanical equiv.of heat app.	04440-00	1
2	Friction cylinder CuZn, m 1.28 kg	04441-02	1
3	Friction cylinder Al, m 0.39 kg	04441-03	1
4	Support rod, stainless steel, l = 250 mm, d = 10 mm	02031-00	1
5	Right angle clamp expert	02054-00	1
6	Spring balance,transparent, 10 N	03065-05	1
7	Spring balance,transparent, 100 N	03065-07	1
8	Digital stopwatch, 24 h, 1/100 s and 1 s	24025-00	1
9	Bench clamp expert	02011-00	1
10	Universal clamp with joint	37716-00	1
11	Commercial weight, 1000 g	44096-70	1
12	Commercial weight, 2000 g	44096-78	1
13	Portable Balance, OHAUS CR5200	48915-00	1

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Setup and Procedure

Setup

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The experimental set-up is shown in Fig. 1. To start with, the friction cylinder and the crank handle are fixed to the rotating bearing and the base plate is clamped very firmly to the table with the screw clamps. Failure to do so may cause tilting of the base plate during cranking, which in turn may cause the thermometer introduced into the bore hole to break. The friction band fastened to the dynamometer is laid 2.5 times around the friction cylinder (so that the dynamometer is relieved when the crank is turned clockwise). The 2 kg weight is attached to the lower end of the friction band. To measure temperature, the thermometer is held with a universal clamp and carefully introduced into the bore hole of the friction cylinder. The thermometer and the bore hole of the cylinder must be carefully aligned, so the thermometer will not be damaged while the cylinder is rotating. To improve thermal contact, the bore hole is filled with heat conducting paste. Weigh the friction cylinders at the beginning of the experiment.

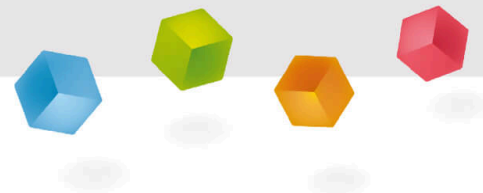
Procedure

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At the beginning of the measurement, temperature is recorded every thirty seconds during four minutes. After this, the crank handle is turned a certain number of times (e. g. 200 times) as fast and regularly as possible. Simultaneously, the force F_D acting on the suspension is determined with the dynamometer. Subsequently, the continuous temperature decrease is noted at intervals of thirty seconds.

The second part of the experiment is carried out similarly, in order to measure the thermal capacity of an aluminium cylinder and that of a brass cylinder whose mass is double the mass of the aluminium cylinder. Before starting the experiment, the friction cylinder and the friction band should be wiped with a dry cloth to rid them from metallic abrasion dust. When using the aluminium cylinder, friction should not exceed 10 N, (for this, the 1 kg weight is used), in order to avoid severe abrasion and soiling of the friction band. The specific thermal capacity of the cylinder is calculated from the number of crank rotations n , the mean force at the dynamometer F_D and the increase of temperature ΔT .

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Evaluation

Results (1/6)

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The thermal capacities of the insulated bearing and of the heat conducting paste were neglected. Fig. 3 displays a temperature-time diagram for an example of measurement. The shape of the graph makes it clear, that thermal energy is continuously given off to the environment while the crank is being turned. Hence, temperature difference ΔT is determined as is generally done for the determination of specific heat: momentary temperature compensation is extrapolated through prolongation of both straight graph branches. If a parallel line to the temperature axis is drawn in such a way that two equal surfaces A_1 and A_2 are formed, the difference of the ordinates of the points of intersection of the two straight lines is the temperature difference ΔT sought after.

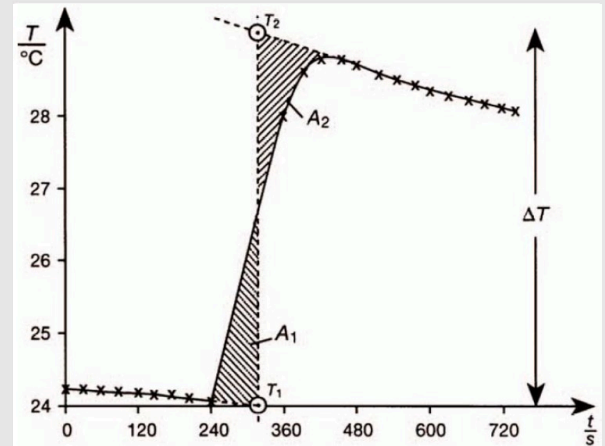


Fig 3: Temperature-time diagram for a measurement example.

Results (2/6)

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The cylinder is cooled mainly due to convection. In this case we can use the Newton's law of cooling as a good approximation: "The rate of heat loss of a body is proportional to the difference in temperatures between the body and its surroundings".

In this approximation the temperature difference in Fig. 3 corresponds to the heat loss rate (first derivative of total heat amount). The integral of temperature difference thus equals to the total amount of lost heat, which graphically corresponds to the area confined under the temperature curve in Fig. 3.

There are two curves in Fig. 3. The first one (real experiment) corresponds to the mechanical work slowly transferred into heat. The second curve (imaginary experiment) shows instantaneous transfer. In both experiments the cylinder has the same amount of heat in the beginning of the experiment and after heating due to friction. Mechanical work is the same for both curves. Thus the amount of lost heat (area under curves) due to convection must be the same for both curves. This is possible only if $A_1 = A_2$, which uniquely defines the position of the vertical line (instantaneous process) in Fig. 3.

Results (3/6)

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Exponential cooling process can be approximated with straight lines for short time intervals.

In the case of a measurement example where the residual load on the dynamometer is $F_D = 3 \text{ N}$, work after 200 rotations of the friction cylinder amounts to:

$$\Delta W = 1301 \text{ J}$$

According to equation (3), the resulting temperature increase of the friction cylinder, $\Delta T = 5.1 \text{ K}$, produces the following amount of thermal energy:

$$\Delta Q = 1296 \text{ J}$$

If the mechanical heat equivalent is calculated, the quotient (1.003) amounts to 1 within the limits of measurement error. As heat is a form of energy, the total amount of mechanical energy must be transformed into an equal amount of heat energy according to the law of conservation of energy. In fact, the mechanical equivalent of heat must thus amount to:

Results (4/6)

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$$\frac{\Delta W}{\Delta Q} = 1 \quad (6)$$

Similar experiments, in which electric energy is converted to thermal energy, show that the corresponding electric equivalent of heat $\Delta W_{\text{el}}/Q$ also amounts to 1. These results are comprehensively summarized in the first law of thermodynamics.

The change of internal energy ΔU of a system is equal to the amount of heat energy ΔQ and to the amount of work ΔW supplied from outside, independently of the supply procedure:

$$\Delta U = \Delta Q + \Delta W \quad (7)$$

If, inversely, equation 6 is assumed ($\Delta W = \Delta Q$), the specific thermal capacity can be determined with this equipment. If equations 3 and 5 are transformed according to the thermal capacity of the friction cylinder, the following result is obtained:

Results (5/6)

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$$C = \frac{\Delta W}{\Delta T} = 8 \frac{\text{J}}{\text{gK}} \quad (8)$$

The specific thermal capacity, defined as the quotient between the thermal capacity of a body C and its mass:

$$c = C/m \quad (9)$$

Is a material constant, which at normal ambient room temperatures and atmospheric pressure depends very little on absolute temperature. It is to be noted from equation 9 that the thermal capacity of a homogeneous material is strictly proportional to its mass.

Results (6/6)

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An example measurement with a rest load $F = 4 \text{ N}$ on the dynamometer, 200 turns of the crank, and a temperature increase of $\Delta T = 2.5 \text{ K}$ yielded a thermal capacity of

$$C = 501 \text{ J/gK}$$

for a brass cylinder of 1280 g. (that is actually about twice the thermal capacity of a cylinder of half the weight (247 J / gK)). The specific thermal capacity calculated with equation 9 is:

$$C_{\text{brass}} = 0.392 \text{ J/gK}$$

The measurement of the aluminium cylinder yielded the following value:

$$c_{\text{al}} = 0.870 \text{ J/gK}$$

as compared to the theoretical value $c_{\text{al}} = 0.902 \text{ J/gK}$ for the specific thermal capacity of aluminium at room temperature.